

Pipe Stress Analysis And Sizing

# in 4.84 version





#### PASS/START-PROF



#### Quick Pipe Stress Analysis & Optimal Sizing

#### Presenter:

Dr. Alex Matveev START-PROF Product Owner Development, Training, Support of START-PROF Since 2005

matveev@passuite.com

#### LinkedIn: linkedin.com/in/alex-matveev/



About one year has passed since our last version of PASS/START-PROF Software release.

We worked hard during this time.

Check out that we have prepared for our customers!



### PASS/Start-Prof | External Interfaces

- Added export & import from AVEVA E3D version 3.1
- Added export & import from AVEVA MARINE version 12.1SP4 and 12.1SP5
- Added AVEVA MDS Support

PIPING AND EOUIPMENT

ANALYSIS & SIZING SUITE

• Added import from Autodesk Revit to PASS/START-PROF





#### PASS/Start-Prof | Codes Updates

- Updated code ASME B31.9-2017 Building Services Piping (USA)
- Updated code ASME B31.4-2019 Pipeline Transportation Systems for Liquids and Slurries (USA)
- ISO 14692-2017 Glass-reinforced Plastics Piping (GRP). Updated the database



#### PASS/Start-Prof | Codes Updates

Ring bending stress is calculated using finite element method with geometrical nonlinearity and consider stiffening effect of the pressure

7.8 Allowable stresses

The sum of the hoop stresses shall be defined by the following formulae:



- ASME B31.12-2014 Hydrogen Piping and Pipelines (USA)
- BS PD 8010-1:2015 Pipeline systems Part 1: Steel pipelines on land (UK)
- BS PD 8010-2:2015 Pipeline systems Part 2: Subsea pipelines (UK)
- CSA Z662 + Ch.11 Oil and gas pipeline systems (Canada)
- GOST R 55989-2014 Gas and Oil Transmission Pipelines for Pressure Greater 10 MPa
- GOST R 55990-2014 Field pipelines (Russia)
- SP 284.1325800.2016 Field pipelines (Russia)
- SP 33.13330.2012 Steel Pipelines (Russia)
- Individual databases was created and filled with Material properties for all new codes

Materials	-	□ ×	Materials	5 <del>3</del> 8	×											
Select Material			Select Naterial		Show		Material:	CSA Z245 G	ade 241	Class: C	arbon or L	ow Alloy Steel			5	□ ×
Code Manufacturing method Material	Code         RD 10-249-98 (Power piping, Russia)           Aanufacturing method         ASME B31.12 PL (Hydrogen Piping and Pipeline ASME B31.12 PL (Hydrogen Piping and Pipeline ASME B31.12 PL (Hydrogen Piping USA)           Vlaterial         ASME B31.32018 (Process piping, USA)           ASME B31.52016 (Refrigeration Piping and He ASME B31.52016 (Refrigeration Piping and He ASME B31.52016 (Gas Transmission, USA)           ASME B31.52017 (Building Services Piping, USA)           ASME B31.52016 (Gas Transmission, USA)           ASME B31.52017 (Building Services Piping, USA)           ASME B31.52016 (Gas Transmission, USA)           ASME B31.52017 (Building Services Piping, USA)           CSA 2662-19 (Oil and gas pipeline systems, Ca DUC 1 5396-2014 (Power piping, Cinna)           EN 13480-2017/EN 13941-2019 (Metallic Indu Car Part Part Part Part Part Part Part Pa	Add Copy To Delete Export Import	Code Manufacturing method Material	CSA 2962-19 (DI and gas pipeline systems, Ea CSA 2245 Grade 305 A139 A A139 A A139 B A139 B A504 H A504 H A504 H A504 H A508 B A49 B A50 B A49 B A49 B A49 B A50 B A49	Add Copy To Delete Export Import		Databas Units and Data source Densty	e can only be set as MPa f CSA 7830	edited if datal ar ASME -kal 1245	base files are o ) kg/m3	pen for editing	g and if stress				
	DDT 5356-2014 (rower piping, China) EN 13480-2017/EN 13941-2019 (Metallic Industrial GB 50253-2014 (Oil Pipeline, China) GB 50316-2008 (Metallic Industrial Piping, China) GB/T 20801-2006 (Process Piping, China) GOST 32388-2013 (Process piping, Russia) GOST 34233.1-2017 (Steel vessels) GOST 455596-2013 (District heating piping system	Help .::		AP1-82, X42 AP1-52, X45 AP1-52, X55 AP1-52, X55 AP1-52	Heb		Temperature *C 20 120	Vield Stress (Sy), kgf/sq.cm 2409.9934 2409.9934	Elastic Modulus kgf/sq.cm 2070000 2070000	Expansion Coeff, 1/°C 1.12e-005 1.12e-005 1.12e-005	Poisson's Ratio (v) 0.3 0.3 0.3	3			•	Add Delete Prrt
	ISO 14692 (GRP piping systems, international) Plastic Piping Systems (HDPE, PP, PVC) RD 10-249-98 (Power piping, Russia) RD 10-400-01 (District heating piping systems, Russ RTM 38.001-94 (Process piping, Russia) SNP, SP, GOST (Gas & oil transmission piping systems) STO 91579448-01.1-2013 (GRP piping systems)			CSA 2245 Gaske 355 <u>CSA 2245 Gaske 355</u> <u>CSA 2245 Gaske 414</u> <u>CSA 2245 Gaske 414</u> <u>CSA 2245 Gaske 420</u> <u>CSA 2245 Gaske 420</u> <u>CSA 2245 Gaske 550</u> <u>CSA 2245 Gaske 550</u> <u>CSA 2245 Gaske 550</u>		NIZ <sup>I</sup>	150	2337.6936	2070000	1.12e-005	0.3		Save	ок	Cancel	Help

EN 13941-2019 District heating pipes - Design and installation of thermal insulated bonded single and twin pipe systems for directly buried hot water networks







Calculate And Check Stresses In Polyurethane Insulation (EN 13941 7.3.1, 7.3.2, EN 253). Check Stresses from Surface Vehicle Loads



- START-PROF has everything that is needed for district heating networks analysis and widely used for it since 1998.
- Pre-heating analysis
- Single use compensators analysis + Database + Distance Calculation





- Added LOGSTOR, POWERPIPE, +GF+ Urecon Polyurethane Pre-insulated Pipe Jacket Sizes
   Database for district heating and district cooling networks
- Now it is allowed to add expansion cushions on the vertical pipes

5	GRUN.ctp [GOST R 55596-2013] - PASS	5/Statt-Prof 2018 v.04.84 #1 - (GRUN.ctp)	
📙 Edie Edie View Nevigetion Graphic	a Joseft Toola Service Dytabases Analysis Oytpu	ut Window Help	- 0 X
	A to man work 1. Main mode	· A 19	e
a & a a a a - a a a t a B	PD. TKY YROAUL	PREX-1	PUR Insulation Jacket Properties
operties # × / A Trub	datal I I Input O		
lpe (2-12) +			Codes
21		🚰 Pipe Properties 🛛 🗙	
3 Main		Per 212 Company	
Trant Node 2		E rues baev	FOCT 30732-2006 Change Series 1 Change
End Node 12		Nate	Chinese National Standard Series 2
Name		Main Additional Soll	LOGSTOR Add Series 3
Input Type Projections		Properties	POWERPIPE 200
a Projections/s 0 m, 0 m, 1.5 m		Outer Casing Dameter 000 min	+Gi++Urecon Delete Delete
A Diameter x Tr 630 mm X 10 mn		Casing Walt Truckness 10 mm	
Pipe Material 20			×
Comprise All Dimm		Consider Soil Movements No.	T Diameter And Thickness of the Jacket
Pressice MP. 16 MPa			
Test Pressure 2 MPa		Statt Rode (2)	Type Do mm Do mm Di mm Th mm
Temperature, 140 °C		Automatic transmission	
Uniform Wei; No, 0.153 tf/m, N		Lingthing the pipe skill 1.5 m	Series 1 26.9 20 90 3 Delete
1 Additional			Series 1 33.7 25 90 3
Weld Guality 1.00			5 Series 1 42.4 22 110 2
Weld Guality 0.80		End Node (12)	
Weld Quality 0.90	10		- Series 1 48.3 40 110 3
3 Soil		Depth to the Pipe Aux 0 m	Series 1 60.3 50 125 3
Outer Casing 800 mm			5 · · · · · · · · · · · · · · · · · · ·
Casing thickr 10 mm			series 1 70.1 03 140 5
Contrider cell No		Sel	Series 1 88.9 80 160 3
Poet song T Open Trench		Pau Leying Method Opentrench +	Series 1 114.3 100 200 3.2
Backfill Soll C 01		Backfé Sol Type Of	
Foundation 5 81		Enstein Editor III	Series 1 139.7 125 225 3.4
Insulation typ Polyurethane for		too see the the	Series 1 168.3 150 250 3.6
Insulation adl 0.67		Insulation and Cushions Insulation Tripe Polyanthana from 1	
Cushion pres. Cushions absent		Insulation adhesion factor 0.67	Save Close Help
or and warning messages		Custon Presence Yes +	
Type Node/pipe Description		Help	
Notes Node4 (W305) Tpytionpoed	поднимиется над опорой (Рабочее состояние) - 1. Main m	node' † DK Cancel Nets	

### PASS/Start-Prof | Code Updates

Updated material database EN 13480/EN 13941. Added all piping materials by EN 10216-1-2013, EN 10216-2-2013, EN 10216-3-2013, EN 10216-4-2013, EN 10216-5-2013, EN 10217-1-2019, EN 10217-1-2019, EN 10217-2-2019, EN 10217-3-2019, EN 10217-4-2019, EN 10217-5-2019, EN 10217-6-2019, EN 10217-7-2014, EN 10220-2002 (2007), EN 10253-2-2007. Added automatic material properties selection depending on wall thickness and seamless/welding option





Added Minimum Design Metal Temperature (MDMT) calculation according to 323.2.2 (a), (b), (d), (e), (f), (g), (h), (i), (j) of ASME B31.3-2018. Added into material database. START-PROF calculates the MDMT according to figure 323.2.2A and figure 323.2.2B depending on the calculated stress ratio if user select appropriate option in project settings, taking into account the code requirements 323.2.2 (g), (h), (i).



#### Table A-1 Basic Allowable Stresses in Tension for Metals (Cont'd)

Numbers in Parentheses Refer to Notes for Appendix A Tables; Specifications Are ASTM Unless Otherwise Indicated

									Speci Mi Streng	ified n. th, ksi	Basic Stres at Temj °F [N	Allow s, S, I Meta perati lote (	vable ksi, l ure, [1]]	
	_			Class/				Min.			Min.			
Materia	Spec. al No.	Type/ Grade	UNS No.	Condition/ Temper	Size, in.	P-No. (5)	Notes	°F (6)	Fensile	Yield	Temp. to 100	200	300	
Carbon St	teel — Pipe	es and Tul	bes			(-)		- (5						
A285 Gr. /	A A134					1	(8b)(57)	В	45	24	15.0	14.7	14.2	
A285 Gr. /	A A672	A45	K01700			1	(57)(59)(67)	В	45	24	15.0	14.7	14.2	
Butt weld	API 5L	A25				1	(8a)(77)	-20	45	25	15.0	15.0	14.7	



After analysis the output report table is provided. For each pipe START-PROF show <u>if</u> <u>the impact test is needed or not</u>

							•
📕 Input 🐵 🐻	MDMT 🖸						
Object	Start End node	Thickness, cm	Material	Stress Ratio, r	Tmin, °C	MDMT, °C	Output
Above ground pipe	3,Restrained	0.600	A106 B	0.294	-40	-48	ОК
Above ground pipe	23	0.600	A106 B	0.395	-40	-48	ОК
	5,Bend	0.600	A106 B	0.840	-40	-37.869	Impact Test
Above ground pipe	6,0 Flange	0.600	A106 B	0.436	-40	-48	ОК
	24	0.600	A106 B	0.400	-40	-48	ОК
Above ground pipe	6,0 Flange	0.600	A106 B	0.342	-40	-48	OK
	8	0.600	A106 B	0.373	-40	-48	ОК
Above ground pipe	8	0.600	A106 B	0.317	-40	-48	ОК
	25	0.600	A106 B	0.283	-40	-48	ОК
Above ground pipe	27	0.600	A106 B	0.430	-40	-48	ОК
	9	0.600	A106 B	0.951	-40	-31.783	Impact Test
Above ground pipe	8	0.600	A106 B	0.330	-40	-48	OK

Figure 323.2.2A Minimum Temperatures Without Impact Testing for Carbon Steel Materials (See Table A-1 or Table A-1M for Designated Carve for a Listed Material; see Table 323.2.2A for Tabular Values)

Figure 323.2.28 Roduction in Lowest Exemption Temperature for Storis Without Impact Testing (See Table 123.2.26 for Tabular Values)





#

1(0)

2 (2)

Name

 $\checkmark$ 

OPE

1.1 (0) occ1.1

3 (1) Test mode

Updated material library, weld reduction factors for ASME B31.3-2018.

Added occasional allowable calculation for elevated temperature fluid service 302.3.6 (2) ASME B31.3-2018,

added appendix V. Added "Time duration", "Alternative Occasional" options to operating mode editor. Added Larson-Miller constant "C" into ASME B31.3 material database.

📙 Input 💿 👩 Stress 🖸									
Operating Mode			~	Show E	Equations	Stree			
1.1 'occ1.1' •				Creep S	Stress				
Object	Start End	Prii Si	mary Lo tress, (ks	ads si)	Notes				
	node	SI_Alt	k*Sh	%	]				
Above ground pipe	14	5.012	5.960	84.1					
	29,2 Flange	7.181	5.960	120.5	1				
Forged Elbow	29,2 Flange	8.444	Sh, 4.6	84 ksi					
Above ground pipe	29,2 Flange	7.805	Sy, 18.	616 ksi					
	15	4.266	ti=500	0 hour					
Above ground pipe	14	5.172	C=20						
	16	5.325	Te, 481						
Weldolet (branch welded-on fitting)	16	12.256	S02, 5.960 ksi						
Above ground pipe	16	3.967	min(49	Sh,0.8*0.9	9Sy,S02),	5.960 ksi			
	20.1.0	6 220	5.000	1015	4				



Added automatic creep-rupture usage factor calculation according to ASME B31.3-2018 Appendix V (V303.1-V303.3)

#### V303.2 Determine Creep-Rupture Usage Factor

The usage factor, u, is the summation of individual usage factors,  $t_i / t_{ri}$ , for all service conditions considered in para. V303.1. See eq. (V4).

 $u = \sum \left( t_i / t_{ri} \right)$ 

#### (V4) V303.3 Evaluation

#### where

- *i* = as a subscript, 1 for the prevalent operating condition;
   *i* = 2, 3, etc., for each of the other service conditions considered
- $t_i$  = total duration, h, associated with any service condition, *i*, at pressure,  $P_i$ , and temperature,  $T_i$
- $t_{ri}$  = as defined in para. V303.1.4



	ACME DO1	2 2010 Cather	(C < 0-2)					
	7000 4100	.3-2010 Calbor	1(000)					
Density	/833.41303		3	🗹 Maxir	num f=1.2			
Larson-Miller constant C	20	-						
Creep Factors	Carbon, Sil	In Steel						-
F	Stress (S), ksi	Stress (Sy), ksi	Modulus ksi	Coeff. 1/F	Ratio (v)	WI		Delet
-325	16	30	31400	5.5e-006	0.292	1.0		
-200	16	30	30800	5.79e-006	0.292	1.0		
-150	16	30	30300	5.9e-006	0.292	1.0		
	16	30	29891	6.2e-006	0.292	1.0		
-50		30	29400	6.4e-006	0.292	1.0		Print
-50 70	16			C 47 00C	0.202	10		
-50 70 100	16 16	30	29262	0.4/e-000	0.292			



Added pipe wall thickness calculator and bend wall thickness calculator for all new codes like ISO 14692-2017, EN 13941, ASME B31.12, BS PD 8010, CSA Z662, GOST R 55989, GOST R 55990, SP 284.1325800.2016, SP 33.13330.2012.

All pipes and fitting wall thicknesses are automatically checked before every run of the pipe stress analysis according to the selected code





Added ability to enter the custom hanger allowable rotation angle for different types of hangers. PASS/START-PROF automatically check the hanger rotation angle and show the note message after analysis if restriction is violated

Automatically Insert	Caps in Free Er	nd Nodes			
Anchor Coefficient by De	efault	1			
Number of Cycles EN 13	941	Hanger Allowable R	otation, °		
Transmission Pipelines:	100	Spring Hanger:	5		
Distribution Pipelines:	250	Rigid Hanger:	5		
House connections:	1000	Constant Hanger:	0		
Piping Size Specification	ANS	I 🔹		•	
PUR Insulation Jacket Co	ode LOG GOS Chin LOG POW +GF HTN	STOR T 30732-2006 9 ese National Sta STOR /ERPIPE + Urecon			



- Added automatic local pipe wall buckling check per ASME B31.8-2018
- EN 13941-2019 7.2.4.2



🖉 🖸 🕞 🔁	Stress	🕑 🛛 👩 Stał	oility 🖸 🗌					
Operating Mode								
1 操作模式'(0)		• ?						
Object	Start End	C1 Local B Hot Condit	Buckling in tion, (MPa)	C1 Local B Cold Condi	uckling in tion, (MPa)	C1 Local B Test Condit	Buckling in tion, (MPa)	Notes
	node	calcu- lated	allow- able	calcu- lated	allow- able	calcu- lated	allow- able	
Buried pipe	1	36.09	33.88	38.39	33.88	0.01	33.88	1,2
	2	12.95	33.88	34.84	33.88	0	33.88	2
Bend	2							
Buried pipe	2	21.44	33.88	34.63	33.88	0	33.88	2
	3	35.93	33.88	38.89	33.88	0.01	33.88	1,2

- For  $(D_o t_n)/(2t_n) \le 28.7$ 
  - $S_{st} = 0.0016E$
- For  $(D_o t_n)/(2t_n) > 28.7$  $S_{st} = (0.0458 \cdot 2t_n/(D_o - t_n) + 0.00003)E$

Added built-in calculator in some input fields

Pipe Properties			×	🚰 Pipe Properties		×
ipe 1-2	Pipe is Bu	uried		Pipe 1-2	Pipe is Bur	ied
ame				Name		
Main Additional Soil				Main Additional So	oil	
projections -				projections Projections	•	
Pipe Length	1	m		Pipe Length	7	m
DX	1+2*3	m		DY	7.000000	m
DY	0	m		DY	0	m
DZ	0	m		DZ	0	m
Properties				Properties		
Outer Diameter	219	mm		Outer Diameter	219	mm
Wall Thickness	6	mm		Wall Thickness	6	mm
Mill Tolerance	12.5	%		Mill Tolerance	12.5	%
Corrosion Allowance	0	mm		Corrosion Allowance	0	mm



- Updated database ASME B36.10M-2018
- Added more than 140 new standards into pipes, tees, bends, and reducers database, including ASME B16.9 and lot of Russian GOST, OST, RD, TU codes

<u>Materials Library</u>	Manufacturing Technology	Manufacturing Type	Standard	Assortment	Schedule	NPS, in	Nominal Diameter	Material		Size	Diameter, cm	Thickness, cm	Mill- Tolle- rance,	Veight, S kg	tandard Group			Add				
i fanabie <u>o</u> pringsm													cm					Delete				
<u>Constant Springs</u>	<pre>vent cet&gt;</pre>	<pre>v v v v v v v v v v v v v v v v v v v</pre>	×	• <b>*</b>	105	1/9		( not cot)		6. NDS 1/9. SCH 105	1 029702	• 12446	0 0	0								
Soils	<not set=""></not>	<not set=""></not>	ASME B36.10M-2018	A EL-	105	1/0	6	<not set=""></not>	DN		1.020702	0.12440	0 0	~								
<u>So</u> lishi	(not set)	(not set)	ASME B36.19M-2004	et >	805	1/8	6	<not set=""></not>	DN	L Tees												
Insulation	(not set)	<not set=""></not>	FOCT 10705-80 rp. B	et.	10	1/0	6	<not set=""></not>	DI	2000												
e	<not set=""></not>	<not set=""></not>	ГОСТ 10706-76 гр. В	et /	20	1/0	6	<not set=""></not>		Type weldrig				1								1
Expansion Joints	<not set=""></not>	<not set=""></not>	FOCT 10706-76*	et.>	40	1/0	6	<not set=""></not>		March Same	and a second	in the second	-	Header	Brench	Header DN.	Branch DN.	Header NPS.	Branch TVPS,	Sale.	Header	Branc
Pines	<not set=""></not>	<not set=""></not>	FOCT 20295-85	er>	40	1/0	6	Knot sets		Manufacturing technology	Standard	evtatorna)	-308	Com Com	.cm	cm	om	34	in	Schedule	cm.	Em
<u>-</u> ipesiii	<not set=""></not>	<not set=""></not>	ГОСТ 20295-85 тип 2	et.>	310	1/0	6	<not set=""></not>			Y			-	-	-	ŀ	-		-		
Bends	<not set=""></not>	<not set=""></not>	ГОСТ 20295-85 тип 3	et 2	ou VC	1/0	6	Knot sets		«not set»	Remove Fitter	*	21,3-13.7	2.13	1.37	15	8.0	1/2	1/4	120	0	0
-	<not set=""></not>	<not set=""></not>	FOCT 3262-75*	et>	160	1/0	6	<not set=""></not>		<not set=""></not>	ASME 816.9-2012 FOCT 17376-2001		21,3-13,7	2.13	1.37	1.5	6.8	1/2	1/4	140	0	0
lees	<not set=""></not>	<not set=""></not>	FOCT 32678-2014	et>	100	1/8	0	<not set=""></not>		<not set=""></not>	OCT 108.104.04-82 OCT 108.104.05-82		21.3-13.7	2.13	1.37	13	0.8	1/2	1/4	100	8	0
Reducers	<not set=""></not>	<not set=""></not>	FOCT 550-75*	et>	XX5	1/8	0	<not set=""></not>	Dr	<not set="&lt;/td"><td>OCT 108.104.06-82</td><td></td><td>21.3-13.7</td><td>2.13</td><td>1.37</td><td>1.5</td><td>0.8</td><td>1/2</td><td>1/4</td><td>XXS</td><td>0</td><td>0</td></not>	OCT 108.104.06-82		21.3-13.7	2.13	1.37	1.5	0.8	1/2	1/4	XXS	0	0
<u>Reducers</u>	<not set=""></not>	<not set=""></not>	FOCT 8731-74 rp. B	et>	10	1/4	8	<not set=""></not>	Dr	<not set="&lt;/td"><td>OCT 108-104.07-82 OCT 108-104.08-82</td><td></td><td>21.3-13.7</td><td>2.13</td><td>1.37</td><td>1.5</td><td>0.8</td><td>1/2</td><td>1/4</td><td>55</td><td>0</td><td>0</td></not>	OCT 108-104.07-82 OCT 108-104.08-82		21.3-13.7	2.13	1.37	1.5	0.8	1/2	1/4	55	0	0
Flanges	<not set=""></not>	<not set=""></not>	FOCT 8731-74*	et>	30	1/4	8	<not set=""></not>	DI	<not set=""></not>	OCT 108.104.09-82 OCT 108.104.13-82		21.3-13.7	2.13	1.37	1.5	8.0	1/2	1/4	105	0	0
- 5	<not set=""></not>	<not set=""></not>	FOCT 8733-74 rp. B	et>	40	1/4	8	<not set=""></not>	DI	«not set»	OCT 108-104.14-82		21.3-13.7	2.13	1.37	1.5	0.0	1/2	1/4	20	0	0
<u>G</u> asket	<not set=""></not>	<not set=""></not>	FOCT 9940-81*	et>	STD	1/4	8	<not set=""></not>	DI	<not set=""></not>	OCT 108.104.15-82 OCT 108.104.16-82		21,3-13.7	2.13	1.37	1.5	0.8	1/2	1/4	10	0	0
In sulation, Is shot	<not set=""></not>	<not set=""></not>	FOCT 9941-81	et>	80	1/4	8	<not set=""></not>	DI	<not sets<="" td=""><td>OCT 108 104 17-82</td><td></td><td>21.3-13.7</td><td>2.13</td><td>1.37</td><td>15</td><td>0.8</td><td>1/2</td><td>1/4</td><td>405</td><td>0</td><td>0</td></not>	OCT 108 104 17-82		21.3-13.7	2.13	1.37	15	0.8	1/2	1/4	405	0	0
Insulation Jacket	<not set=""></not>	<not set=""></not>	FOCT 9941-81*	et>	XS	1/4	8	<not set=""></not>	DI	<nut set="&lt;/td"><td>OCT 108.104.19-82</td><td></td><td>21.3-13.7</td><td>2.13</td><td>1.37</td><td>1.5</td><td>0.8</td><td>1/2</td><td>1/4</td><td>40</td><td>0</td><td>0</td></nut>	OCT 108.104.19-82		21.3-13.7	2.13	1.37	1.5	0.8	1/2	1/4	40	0	0
	<not set=""></not>	<not set=""></not>	CTO 56517011-042-2010	et>	160	1/4	8	<not set=""></not>	DI	<not sat=""></not>	OCT 108.104.20-82 OCT 108.720.01-82		21,3-13.7	2.13	1.37	1.5	0.8	1/2	1/4	30	0	0
	<not set=""></not>	<not set=""></not>	TY 13.03-011-00212 179-200	3 et>	XXS	1/4	8	<not set=""></not>	DI	Knot set>	OCT 108.720.02-82		21.3-13.7	2.13	1.37	1.5	6.8	1/2	1/4	80	Ø.	0
	<not set=""></not>	<not set=""></not>	ТУ 1301-039-00212179-2010	et>	105	1/4	8	<not set=""></not>	DI	<not set=""></not>	OCT 108.720.04-82		21.3-13.7	2.13	1.37	1.5	0.8	1/2	1/4	60	0	0
	<not set=""></not>	<not set=""></not>	TV 1303-002-08620133-01	et>	405	1/4	8	<not set=""></not>	DN	<not set=""></not>	OCT 108.720.05-82 OCT 108.720.06-82		21.3-13.7	2.13	1.37	1.5	0.8	1/2	1/4	805	0	0
	- not set		Ty 14-3-1080-81		105		Ŭ.	-not set		<not set=""></not>	OCT 108.720.07-92		21.3-15.7	213	1.77	1.3	1	1/2	2/2	141	0	0
	Only first 100	rows are shown	ТУ 14-3-1128-2000						_	<not set=""></not>	ОСТ 36-23-77 СТО ЦКТИ 720.01-20	»	21.3-17.3	2.13	1.73	13	1	1/2	3/8	160	0	10
	To see other r	ows please use filte	ТУ 14-3-1128-82	~						<not sate<="" td=""><td>СТО ЦКТИ 720.02-20</td><td>2</td><td>21.3-17.3</td><td>2.13</td><td>1.73</td><td>1.5</td><td>1</td><td>1/2</td><td>3/8</td><td>120</td><td>0</td><td>0</td></not>	СТО ЦКТИ 720.02-20	2	21.3-17.3	2.13	1.73	1.5	1	1/2	3/8	120	0	0
					_					Knot sets.	СТО ЦКТИ 720.07-20	10	21.3-17.3	2.13	1.73	1.5	1	1/2	3/8	XXS	0	0

Added polypropylene "PP-B" into material database. Data taken from DVS 2205, EN 1778, and DIN 8078

Pipe Properties × Pipe 41-42 Pipe is Buried Name Main Additional cylinder Ŧ Cylindrical Length in XY plane 3.5 m DZ 8.881784e-(m - 45 Angle With X Axis Angle With Y Axis + 45 Properties 710 Outer Diameter mm 🖀 Project Settings... - plastic.ctp × Wall Thickness 42.1 mm General Additional Seismic Wind, Snow, Ice Other Dynamic Pipe Material PE 100 PE 100 PE 67 15-02-2017 Description 1 Date Forces PE 80 Piping Type: Pressure PE-RT type 1 PE-RT type 2 Temperature All Ŧ PP-B PP-H Stress Analysis Code: PP-R Test Pressure PVC-C type 2 Plastic Piping Systems(HDPE, PP, PVC) • PVDF Uniform Weight



Added seismic wave propagation analysis for underground pipelines. START-PROF calculate stress and strain in buried pipeline caused by seismic wave propagation and check the stress and strain limits according to

- ASCE 2001 Guidelines for the Design of Buried Steel Pipe (American Lifelines Alliance). Improved by START-PROF authors, added shear wave effect
- GB 50032 (China)
- GB 50470 (China)
- SNiP 2.05.06-85 (Russia)
- SP 36.13330.2012 (Russia)
- GOST R 55989-2014 (Russia)
- GOST R 55990-2014 (Russia)
- SP 284.1325800.2016 (Russia)
- SP 33.13330.2012 (Russia)







Every pipe branch, turn or anchor cause great axial and bending stresses







Added strain check according to ASCE 2001 Guidelines for the Design of Buried Steel Pipe (American Lifelines Alliance), SNiP, SP, GOST, GB Codes

🖉 🖓 İnput 🐵 🔂 Stre	ess 🖸								
Operating Mode				Show E	Equations	Stress R	lange from Op	peration t	o Cold
1.1 'Soil Seismic Wave Pro	paga 🔻					Add Axia	al Force and	Torsion S	tress ?
Object	Start End	Buried pipir	ng Seismic Cl	heck, (MPa)	Buried pip	ing Seismic	Check, (%)	Notes	
	node	SI	Allow	%	Σ	Allow	%		
Buried pipe	9	515.60	965.27	53.4	0.2009	0.2939	68.4		
	3	515.58	965.27	53.4	0.2009	0.2939	68.4		
Buried pipe	8	510.81	965.27	52.9	0.1985	E, 201051.1	2 MPa		
	9	515.60	965.27	53.4	0.2009	[εa]=0.75(0	.5t/D-0.002	5+3000(P	PD/(2Et)) <sup>2</sup> ), 0.002939
Buried pipe	7	490.91	965.27	50.9	0.1884	[ɛa]%, 0.29	39		
	0	510.01	065.27	52.0	0 1005	0.2020	67 5		

#### Compression strain limit



$$D' = \frac{D}{1 - \frac{3}{D}(D - D_{\min})}$$

🖉 🖓 İnput 🔞 🖉 🔂 Stre	ess 🖸																									
Operating Mode						$\checkmark$	Show	Equation	IS	Stress	s Range f	rom Opera	tion to (	Cold												
Maximum	•					_				Add A	xial Force	e and Tors	ion Stre	ss	?											
Object	Start End node	s	Hoop tress, (M	IPa)		Stres	Prim is in H	ary Loa lot State	ds e, (MPa)			Prir in	mary& Loads Hot Sta	Seconda Stress ite, (MPa	у )		Expansio	n Stress Rar	ige, (MPa)	Buried pipir	ng Seismic C	heck, (MPa)	Buried pip	ing Seismic	Check, (%)	Notes
		Sh	F*E*Sy	%	Seq	F*Sy	%	SI	F*Sy	%	Seq	F*Sy	%	SI	F*Sy	%	Se	Sa	%	SI	Allow	%	Σ	Allow	%	
Buried pipe	9	85	173.75	48.9				24.62	180.99	13.6	154	217.18	70.9	69	217.18	31.8	93.61	217.18	43.1	515.60	965.27	53.4	0.2009	0.2939	68.4	
	3	85	173.75	48.9				24.62	180.99	13.6	154	217.18	70.9	69	217.18	31.8	93.61	217.18	43.1	515.58	965.27	53.4	0.2009	0.2939	68.4	
Buried pipe	8	85	173.75	48.9				24.69	180.99	13.6	153.55	217.18	70.7	68.55	217.18	31.6	93.23	217.18	42.9	510.81	965.27	52.9	0.1985	0.2939	67.5	
	9	85	173.75	48.9				24.62	180.99	13.6	154	217.18	70.9	69	217.18	31.8	93.61	217.18	43.1	515.60	965.27	53.4	0.2009	0.2939	68.4	
Buried pipe	7	85	173.75	48.9				24.98	180.99	13.8	151.70	217.18	69.9	66.70	217.18	30.7	91.60	217.18	42.2	490.91	965.27	50.9	0.1884	0.2939	64.1	
	8	85	173.75	48.9				24.69	180.99	13.6	153.55	217.18	70.7	68.55	217.18	31.6	93.23	217.18	42.9	510.81	965.27	52.9	0.1985	0.2939	67.5	
Long Radius Pipe Bend	2	85	173.75	48.9				35.27	180.99	19.5	204.13	217.18	94.0	119.13	217.18	54.9	84.97	217.18	39.1	732.51	965.27	75.9	0.1589	0.2939	54.1	
Buried pipe	2	85	173.75	48.9				30.03	180.99	16.6	169.99	217.18	78.3	84.99	217.18	39.1	84.91	217.18	39.1	568.23	965.27	58.9	0.1588	0.2939	54.0	
	7	85	173.75	48.9				24.98	180.99	13.8	151.70	217.18	69.9	66.70	217.18	30.7	91.60	217.18	42.2	/00.01	965.27	50.9	0 1884	0 2030	64.1	J

Landslide, Soil subsidence, frost heaving, Permanent ground deformation (seismic fault crossing) can also be modeled. The pipeline strain check is made according to ASCE 2001 (ALA) and GB 50470



The pipeline strain check is made according to ASCE 2001 (ALA) and GB 50470

(U) Operating	Mode					7		1.00	1.00	SUS	✓ 1-1A	× ?	Ad
(2) Landslide								1.00	1.00	ASCE 2001 (ALA)	~	~ ?	Dele
(1) Test										SUS		?	Up
										OCC			Dov
								the second second second second		ASCE 2001 (ALA)			
										GB 50470			
2 'Landslide' (	2)		▼ 1'操作	F模式'(0)	) (2-1A)		•	Show Equations	Add Axial I	Force and Torsion	Stress ?	]	
Object	Start	Landslid	andslide strength, (MPa)			ide st	rength (T	ension Area), (%)	Landslide stre	ength (Compres	sion Area), (%)		
	End node	SI	Allow	%	٤		Allov	v %	٤	Allow	%		
	1	48.20	1930.53	2.5	0.023	87	2	1.2	-0.02386	2	1.2		
Buried pipe	2	186.69	1930.53	9.7	0.092	47	2	4.6	-0.02027	2	1.0		
Buried pipe		77.07	1930.53	4.0	0.038	817	2	1.9	0.03403	0.7304	4.7		
Buried pipe Buried pipe	2		1930.53	2.4	0.022	27	0.730	4 3.1	-0.0227	E, 201906.18 N	IPa		
Buried pipe Buried pipe	2 3	45.84								[sal=0.5t/D-0.0	0025+3000(PD/	((2E+)) <sup>2</sup> 0.007	304
Buried pipe Buried pipe	2 3	45.84	1556155							[[[[]]]=0.50,0,0,0,0,0	002010000(10)	(224)), 0.001	
Buried pipe Buried pipe	2	45.84	1550155							[ɛa]%, 0.7304	50251 5000(1 2)	(220)), 0.001	

- Added ability to specify different seismic anchor movement values for the same phase group. It is used to define various restraint movement on several floors of the same building
- Added new functions to Operation Mode Editor: Disable overload factors, add a factor to weight loads



Added Insulation Joint (Insulation Kit) stress analysis. The axial stress and stress from torsion moment is checked automatically







- Added new object "Cylindrical Shell", that is used for modelling of boilers, pressure vessels, columns, horizontal vessels,
- Added new object "Nozzle", allows to automatically model nozzles of boiler and pressure vessels, and columns. Automatically model shell flexibilities using WRC 297/PD 5500, custom values or FEM method, movements due to thermal expansion of the vessel, checks allowable loads, automatically checks stresses using WRC 107/537/297



TNode Object Properties		×	Mode Object Properties			×		
Vessel Nozzle WRC107/537/297/FEM	1		Vessel Nozzle WRC107/537/297/F	EM				_
Node Node		20	Node .			22		
Material	20		Material	20	•			
Manufacturing Technology	Searchest		Manufacturing Technology	Searchea	•			
Temperature of Vessel	L 100	. TC	Temperature of Vessel	L 100	°C			
Remove Restaints for Hanger Selection	a.		Remove Restraints for Hanger Selec	tion				
Vertical •			Vetical •					
Inside Diameter of Vessel, Di	2000	mn	Inside Diameter of Vessel, Di	2000	mm			
Longth From Anchor to Nozzle, X	0		Length From Anchor to Nozzle, X	0	m			
Length Hom Anchor to Nozze, 1	2		Length Hoti Anchor to Nozzle, 1	2	m			-
					1.44		Versel Netzle WRC107/527/207/EEM	
Vessel Area X +			Vessel Axis X +				Vessel Nozzle WKC107/357/297/PEM	
Nozde Flexibility	Alowable Loads	a / Stresses	Nozde Rexbility	Aloweble Loade	/ Stresses		Tank Nozzle API 650	
By WRC 297	Stress Check by	WRC 297/10 +	By WRC 297	Manually			Pump Nozzle API 610/ISO 13709	
Rigid Manually		-		No Check Manualy			Pump Nozzle ISO 9905	
By PD 5500			Lrad 2.68344709331 mm/kgl	Stress Check by Stress Check by	WRC 297/107 FEM		Bump Nozzle ISO 5199	
By FEM Lor 0 mmAid			Loir 0 mm/kgf	Allowable Loads	by FEM kor			
Liong () mm/kat			Liong 0 em/kgl	Flong 1	kat		In-Line Pump Nozzle API 610/ISO 13/09	
Red 0 today			Bad o tada	Mred 4	inter .		Other Pump Nozzle	
Bet 27722 Apr 14			Ret 27722 OV	Mor c	1901		Turbine Nozzle NEMA SM 23/API 611/API 61	2
Plane 4 2002 - 005 - 14 d			Rear 1 2002 - 000 14 d	Mong c	kgron		Compressor Nozzle API 617/API 619/ISO 10/	20
1444 1.70036-006 AUG CII			rearry 1,20039-006 Akgron	100000-000	101			55.
	-	1750 IV					Air Cooler Nozzle API 661/ISO 13706	
OK	Cancel	Help	OK	Cancel	Hep		Fired Heater API 560/ISO 13705	



Added new object "Tank Nozzle API 650", allows to automatically model the storage tank nozzles. Automatically model flexibilities using API 650, thermal movements of the nozzle, movements and rotation due to tank bulging effect using API 650, tank settlement, automatically checks allowable loads using API 650 and STO-SA 03-002-2009



- Added new object "Pump API 610/ISO 13709", allows to automatically model the pumps, consider thermal movements of the nozzles, checks allowable loads using API 610 and ISO 13709 a) The individual component forces and moments acting on each pump nozzle flange shall not exceed the range specified in Table 5 (T4) by a factor of more than 2.
- Added new object "Pump ISO 9905"
- Added new object "Pump ISO 5199"



- b) The resultant applied force (F<sub>RSA</sub>, F<sub>RDA</sub>) and the resultant applied moment (M<sub>RSA</sub>, M<sub>RDA</sub>) acting on each pump-nozzle fange shall satisfy the appropriate interaction equations as given in Equations (F.1) and (F.2):

$[F_{\rm RSA}/(1,5\times F_{\rm RST4})] + [M_{\rm RSA}/(1,5\times M_{\rm RST4})] < 2$	(F.1)
$[F_{RDA}/(1,5\times F_{RDT4})] + [M_{RDA}/(1,5\times M_{RDT4})] < 2$	(F.2)

c) The applied component forces and moments acting on each pump nozzle flange shall be translated to the centre of the pump. The magnitude of the resultant applied force, FRCA, the resultant applied moment, MRCA, and the applied moment shall be limited by Equations (F.3) to (F.5). (The sign convention shown in Figures 21 through 25 and the right-hand rule should be used in evaluating these equations.)

$F_{\text{RCA}} < 1,5(F_{\text{RST4}} + F_{\text{RDT4}})$	(F.3)
M <sub>YCA</sub>   < 2,0(M <sub>YST4</sub> + M <sub>YDT4</sub> )	(F.4)
$M_{\rm RCA} < 1,5(M_{\rm RST4} + M_{\rm RDT4})$	(F.5)

 $F_{RCA} = [(F_{XCA})^2 + (F_{YCA})^2 + (F_{ZCA})^2]^{0.5}$ 

where	
$F_{XCA} = F_{XSA} + F_{XDA}$	where
$F_{YCA} = F_{YSA} + F_{YDA}$	$M_{\rm XCA} = M_{\rm XSA} + M_{\rm XDA} - [(F_{\rm YSA})(zS) + (F_{\rm YDA})(zD) - (F_{\rm ZSA})(yS) - (F_{\rm ZDA})(yD)]/1\ 000$
$F_{ZCA} = F_{ZSA} + F_{ZDA}$	$M_{\rm YCA} = M_{\rm YSA} + M_{\rm YDA} + [(F_{\rm XSA})(zS) + (F_{\rm XDA})(zD) - (F_{\rm ZSA})(xS) - (F_{\rm ZDA})(xD)]/1\ 000$
$M_{\rm RCA} = [(M_{\rm XCA})^2 + (M_{\rm YCA})^2 + (M_{\rm ZCA})^2]^{0.5}$	$M_{ZCA} = M_{ZSA} + M_{ZDA} - [(F_{XSA})(vS) + (F_{XDA})(vD) - (F_{YSA})(xS) - (F_{YDA})(xD)]/1000$

🛛 🖓 Input 🐵 🔂 Equip	ment 🖸												
Operating Mode	Load	d Case			Show Equations								
1 'main mode' (0)	• Ope	erating W+P+T	•						?				
Object	Start End node	Туре	DN, mm	Frad, N	Fcir, N	Flong N	FR, N	Mrad, N∙m	Mcir, N·m	Mlong, N∙m	MR, N∙m	Sum	Notes
Pump API 610/ISO 13709	Node (1)	Suction, Side	200	-7333	5887	-29592	31050	-2626.53	18306.88	4598.20	19057.39	2.84	1
				9780	6220	7560	6920	3520	5160	7060	4710		
	Node (3)	Discharge, Side	200	1440505	-173	0	1440505	0		28.89	28.89	69.39	1
				9780	6220	7560	6920	3520	5160	7060	4710		
		Summary Loads		1433173	5714	-29592	1433490	-2626.53	33102.90	7657.21	34078.35		1
							20760	[My_sum]=2	2([MradT1] +	[MradT2])=2*	(1760+1760	)=7040	N·m

Added new object "In-line Pump", allows to automatically model the vertical in-line pumps, consider thermal movements of the nozzles, checks allowable loads using API 610 and ISO 13709

🚰 Node Object Properties 🛛 🗙						σ <sub>p</sub> = (σ/2	$+(\sigma^{2/4}+\tau^{2})$	<sup>2)0,5</sup> < 41						(F.6)
In-Line Pump API 610/ISO 13709						σ <mark>i</mark> = [1,27	$F_{\rm Y}/(D_{\rm o}^2 - D_{\rm i}^2)$	<sup>2</sup> )] + [10 2	$MOD_o(M_X^2 + M)$	$Z^{2}^{0,5}/(D_{0}^{4}-1)$	D <sub>i</sub> <sup>4</sup> )			(F.7)
Code API 610						τ = [1,27	$(F_X^2 + F_Z^2)^{0.5}$	${}^{5}V(D_{0}^{2} - L$	D <sub>i</sub> <sup>2</sup> ) + [5 100D <sub>o</sub>	$(M_{\rm Y})/(D_0^4 -$	D <sub>i</sub> <sup>4</sup> )			(F.8)
Length Weight					Fo	r USC units	s, Equations (	(F.9) to (F	.11) apply:					
400 m 50 N						$\sigma_{\rm p} = (\sigma/2)$	$+(\sigma^{2/4}+\tau^{2})$	<sup>2</sup> ) <sup>0,5</sup> < 5 9	50					(F.9)
Material of Pump 20 -						σ = (1.27		211 + [122]	$D(M^2 + M^2)$	0.51//D 4 _ D 4)				(F 10)
Manufacturing Technology Seamless -						o <sub>l</sub> = [1,27	$P_{Y'}(D_0^ D_1^-)$	_]] + [1221	$D_0(MX^2 + MZ^2)$	$D_0^{(D_0)} = D_1^{(I)}$				(F.10)
						<i>τ</i> = [1,27	$(F_{X}^{2} + F_{Z}^{2})^{0.5}$	$[D_0^2 - L]$	$D_i^2) + [61D_0(M)]$	$(V_{\rm P}))/(D_0^4 - D_1^4)$	)			(F.11)
Temperature of Pump L 100 °C					wh	iere								
Factor for Temperature						$\sigma_{\rm p}$ is	s the principa	al stress, e	expressed in m	egapascals (p	ounds-force per s	square inch);		
Nozzle Loading Eactor 2						$\sigma_{\rm I}$ is	s the longitud	dinal stres	s, expressed in	n megapascals	s (pounds-force p	er square in d	n);	
Pump Center of Gravity Coordinate from Node 3						τ i	s the shears	tress, exp	pressed in meg	apascals (pou	nds-force per squ	are inch);		
DX 0 mm DY 500 mm DZ 0 mm						F <sub>X</sub> i	s the applied	force on t	the X axis;					
Suction Node						F <sub>Y</sub> i	s the applied	force on t	the Y axis;					
1 +						F <sub>Z</sub> i	s the applied	force on t	the Zaxis;					
	📕 Input 🐵 🔂 Equipment	8												
	Operating Mode	Load Case					Shov	w Equatio	ons					
	1 'main mode' (0) 👻	Operating V	V+P+T ▼							?				
Discharge Node	Object	Start End node	Туре	DN, mm	Frad, N	Fcir, N	Flong N	FR, N	Mrad, N∙m	Mcir, N·m	Mlong, N·m	MR, N·m	Sum	Notes
	In-Line Pump API 610/ISO 1370	9 Node (1)	Suction, Side	219	-1200421		-28			47.98			0.00	1
Children of the second s					7560	9780	6220		7060	3520	5160			
	<b>NO</b>	Node (2)	Discharge, Side	219	-1200421		-27			47.98			0.00	1
OK Cancel Help					7560	9780	6220		7060	3520	5160			

Added new object "Compressor API 617/API 619/ISO 10439", allows to automatically model the compressors, consider thermal movements of the nozzles, checks allowable loads using API 617 and ISO 10439

	In SI units:		$F_{e} + 1.04M_{e} \le 40.4D_{e}$				(F.5a)			ER .			
Compressor API 617/ISO 10439	$F_{\mu} = 1.09 M_{\mu} \ge 54.1 D_{\mu}$	(F.ta)	In 110C context						Cent	an a	A. A.		
Name	In U.S. customary (USC) units:		in obcians.					12		22			
Material of Compressor 20 •	2.35.01 (123		$2F_i + M_i \le 462D_i$				(F.5b)				1.00		
Manufacturing Technology Seamless •	$3F_s + M_s = 927D_s$	(F.1b)	where					0					
Temperature of Compressor L 100 °C	F, is the resultant force, Newtons (ib) (see Figure F.1),		F. is the combined resultant of a	niel, sidestream, and disch-	rae forces. New	vions (itr)							
actor for Allowable Loads 1	$F_r = \sqrt{F_r^+ + F_r^+ + F_r^+}$	(F-2)	If is the continued reaction of	the meature in the	discharged mater	unts and m	(enantity)						
Andt Avia X •	M, resultant moment, in Newton-meters (It-b) from Figure F.1;		resulting from forces, Newfor	o-mellers (8-b);	and any second		Lairen a.p.				11.55		39
	$M_{z} = \sqrt{M_{z}^{2} + M_{z}^{2} + M_{z}^{2}}$	(F.3)	D <sub>i</sub> is the dameter (mm (in.)) of sidestream, and discharge	of one circular opening e openings. If the equivale	oual to the tota nt nozzle diam	al areas of the	e miet, er than						and the second s
enter of Compressor Coordinate from Node 3 -	For sizes greater than 200 mm (8 in.), use the following values.		230 mm (9 in.), use a value	of D <sub>c</sub> equal to the following		1244-516							C
JX 0 mm DY 500 mm DZ 0 mm	in Srunts:		In SI units:										
encove Restraints for Hanger Selection	$D_r = \frac{(400 + D_{mm})}{3}$ (nm)	(F.4a)	D. (460 + Equivalent Diseaster) 3 0	(anan)			(F.6a)						
C. Cation Records			In USC units:										
1 Sealer Notes	In USC units:		D <sub>1</sub> (IR+ Equivalent Diameter) 0	in)			(F 60)			3			
* Li Ser Manual Loads	$(16 + D_{-})$	Contraction of the second											
	$D_c = \frac{(16 + D_{am})}{\pi}$ (in.)	(F.45)	The absolute value of the individual com- tolowing.	ponents (Figure F.1) of the	se resultants si	hould not exp	ed the					4	
arthuma Noda	$D_{\rm c} = \frac{(16 + D_{\rm sm})}{\pi} ~({\rm in}) \label{eq:D_c}$ in Si units:	(F.4b)	The absolute value of the individual com following. In SI units:	ponents (Figure F 1) of the	se resultants si	nould not exp	and the					ť	
Nacharge Node	$D_{\rm c} = \frac{(16+D_{\rm sum})}{\pi}~({\rm in.})$ In Si units: $P_{\rm c} + 1.64M_{\rm c} \leq 40.4D_{\rm c}$	(F.4b) (F.5a)	The absolute value of the individual composition of the individual composition of the second state of the	ponents (Figure F.1) of the	se resultants si	hould not exp	and the					ť	
Discharge Node 3 • Set Manual Loads	$D_c = \frac{(16 + D_{max})}{z} (in.)$ In Si units: $P_c + 1.64M_c \le 40.4D_c$ In USC units:	(F.4b) (F.5a)	The absolute value of the individual complotowing: In SI units: $F_c=16.1D_c \qquad M_c=24.6D_r$ $F_g=40.5D_c \qquad M_g=12.3D_r$	ponents (Figure F t) of the	se resultants si	hould not exp	eed the					ť	
Discharge Node	$D_c = \frac{(16 + D_{max})}{z}$ (in.) In Si units: $F_c + 1.64M_c \le 40.4D_c$ In USC units: $2F_c + M_c \le 462D_c$	(F-4b) (F-5a) (F-5b)	The absolute value of the individual complotowing: In SI units: $F_c = 16.1D_c$ $M_c = 34.6D_c$ $F_g = 40.5D_c$ $M_g = 12.3D_c$ $F_c = 32.4D_c$ $M_c = 12.3D_c$	ponents (Figure F t) of the	se resultants o	houid not exp	wed the					f	
Additional Nozale 1	$\begin{split} D_c &= \frac{(16+D_{max})}{z} ~(\text{in.}) \end{split}$ In Si units: $\mathcal{F}_c + 1.64M_c \leq 40.4D_c \end{split}$ In USC units: $2\mathcal{F}_c + M_c \leq 462D_c \end{split}$	(F-4b) (F-5a) (F-5b)	The absolute value of the individual complotoeing: In SI units: $F_c = 16.1D_c$ $M_c = 34.6D_c$ $F_g = 40.5D_c$ $M_g = 12.3D_c$ $F_c = 32.4D_c$ $M_g = 12.3D_c$	ponents (Figure F t) of the	se resultanta si	hould not exp	eed the					£	
Acharge Node	$\begin{split} D_c &= \frac{(16+D_{max})}{z} ~(\text{in.}) \\ &\text{in Si units:} \\ & \mathcal{P}_c + 1.64M_c \leq 40.4D_c \\ &\text{in USC units:} \\ & 2\mathcal{P}_c + M_c \leq 462D_c \end{split}$	(F-4b) (F-5a) (F-5b)	The absolute value of the individual complotoeing. In SI units: $F_c = 16.1D_c$ $M_c = 34.6D_c$ $F_g = 40.5D_c$ $M_g = 12.3D_c$ $F_c = 32.4D_c$ $M_g = 12.3D_c$ (D) Input (Decision of the second secon	ponents (Figure F t) of the	se resultants si	nould not exa	Ed Serve Fr					ť	
Vacharge Node	$\begin{split} D_c &= \frac{(16+D_{max})}{z} \ (\text{in.}) \end{split}$ In Si units: $\mathcal{P}_c + 1.64M_c &\leq 40.4D_c \end{aligned}$ In USC units: $2\mathcal{P}_c + M_c &\leq 462D_c \end{split}$	(F-4b) (F-5a) (F-5b)	The absolute value of the individual complotoeing. In SI units: $F_c = 16.1D_c$ $M_s = 34.6D_c$ $F_g = 40.5D_c$ $M_g = 12.3D_c$ $F_c = 32.4D_c$ $M_g = 12.3D_c$ (D) Input (Equipment (C)) Density Mode Load Case 1 Partonal persons' (D) • Operating	panents (Figure F t) of the e g W+P+T •	se resultants d	nould not exa	eed the ⊡ Show Ee	uations	7			ť	
Discharge Node  Discharge Node  S  Set Manual Loads  Additional Nozzle 1  Set Menual Loads  Additional Nozzle 2	$D_c = \frac{(16 + D_{max})}{z} (in.)$ In Si unit: $P_c + 1.64M_c \le 40.4D_c$ In USC units: $2P_c + M_c \le 462D_c$	(F-4b) (F-5a) (F-5b)	The absolute value of the individual complotowing. In SI units: $F_c = 16.1D_c$ $M_s = 34.6D_c$ $F_g = 40.5D_c$ $M_g = 12.3D_c$ $F_c = 32.4D_c$ $M_g = 12.3D_c$ (D) Input (Equipment O) Denting Mode Load Cas 1 Pational pervise' (D) • Operating Object	e g W+P+T + Start Type access	DN, mm	Frad, N F	eed the	uations N FR, N	7. Mtad, N-m	Mcir, N-m	Meng. Nm	HR, Non.	Sum Note
Alacharge Node	$D_c = \frac{(16 + D_{max})}{2} (in.)$ In Si unit: $P_c + 1.64M_c \le 40.4D_c$ In USC units: $2P_c + M_c \le 462D_c$	(F-4b) (F-5a) (F-5b)	The absolute value of the individual complotency. In SI units: $F_c = 16.1D_c$ $M_c = 34.6D_c$ $F_g = 40.5D_c$ $M_g = 12.3D_c$ $F_c = 32.4D_c$ $M_g = 12.3D_c$ $D_c$ Input <b>Sequences O</b> Density Mode Load Case 1 Padoush percent (0) • Operating <b>Object</b> <b>Compresson API 617/API 6159/ISO 10439</b>	e g W+P+T • Start Type node Node (1) Suction, Top	DN, mm	Fead, IN F	eed the	N FR, N 50052	7 Mrad, Nen -2577.85	Mcir, N-m 11010.25	Meng. Nm 8677.81	H. N.m. 14253.98	Sum Note
	$D_c = \frac{(16 + D_{max})}{2} (in.)$ In Si unit: $P_c + 1.64M_c \le 40.4D_c$ In USC units: $2P_c + M_c \le 462D_c$	(F-4b) (F-5a) (F-5b)	The absolute value of the individual complotency. In SI units: $F_c = 16.1D_c \qquad M_s = 34.6D_c$ $F_s = 40.5D_c \qquad M_s = 12.3D_c$ $F_c = 32.4D_c \qquad M_s = 12.3D_c$ Denoting Mode Load Case 1 Pedonal percent (0) • Operating Object Compressor API 617/API 619/ISO 10439	e g W+P+T Start End Node (1) Suction, Top Node (3) Discharge, Ta	DN, mm 5 200 \$ 200	Frad, IN F -15918 1440505	eed the Show Er cir, N Flong 20072320 -173 0	uations N FR, N 9 30952 1440505	7 Mtad, Nen -257783 0	Mcir, N-m 11010.25	Micing, Nim 8677.81 28.39	MR, N-m 14253.68 28.89	Sum Note 432 1 33134 1
	$D_c = \frac{(16 + D_{max})}{2} (in.)$ In Si unit: $P_c + 1.64M_c \le 40.4D_c$ In USC units: $2P_c + M_c \le 462D_c$	(F-4b) (F-5a) (F-5b)	The absolute value of the inductual completioning. In SI units: $F_c = 16.1D_c \qquad M_c = 34.6D_c$ $F_c = 40.5D_c \qquad M_c = 12.3D_c$ $F_c = 32.4D_c \qquad M_c = 12.3D_c$ Denoting Mode Load Case 1 Pedonal person (ID) • Operating Object Compressor API 617/API 618/ISO 10439	e g W-P+T • Start Type Node (1) Suction, Top Node (3) Discharge, Te ant1	DN, mm 5 250 \$ 200	Frad, N F -15918 1440505	eed the Show Er cir, N Floring 2007 -2320 -173 0	uations N FR, N 9 30062 1440505	7 Mrad, N-m -2577.83 0	Mcir, N-m 11010.28	Micng, N-m 8677.81 28.89	MR, N-m 14253,98 28.89	Sum Note 432 1 33134 1
Alacharge Node	$D_c = \frac{(16 + D_{max})}{2} (in.)$ In Si unit: $P_c + 1.64M_c \le 40.4D_c$ In USC units: $2P_c + M_c \le 462D_c$	(F-4b) (F-5a) (F-5b)	The absolute value of the individual completioning. In SI units: $F_c = 16.1D_c \qquad M_c = 34.6D_c$ $F_c = 40.5D_c \qquad M_c = 12.3D_c$ $F_c = 32.4D_c \qquad M_c = 12.3D_c$ Denoting Mode Load Case 1 Performant Compression API 617/API 618/ISO 10439	e g W-P+T • Start Type Node (1) Suction, Top Node (3) Discharge, Te est2	DN, mm 5 250 \$ 250	Frad, N F -15918 1440505	eed the Show Er cir, N Floing 2007 -2320 -173 0	N FR, N 9 30962 1440505	7 Miad, N-m -2577.83 0	Mcir, N-m 11010.28	Micng, N-m 8677.81 28.39	MR, N-m 14253.98 28.89	Sum Note 430 1
Image: Set Manual Loads       Additional Nozzle 1       Image: Set Manual Loads       Image: Set Manual Loads	$D_{c} = \frac{(16 + D_{max})}{2} (in.)$ In Si unit: $P_{c} + 1.64M_{c} \le 40.4D_{c}$ In USC units: $2P_{c} + M_{c} \le 462D_{c}$	(F 4b) (F 5a) (F 5b)	The absolute value of the inductual completioning. In SI units: $F_c = 16.1D_c \qquad M_c = 34.6D_c$ $F_c = 40.5D_c \qquad M_c = 12.3D_c$ $F_c = 32.4D_c \qquad M_c = 12.3D_c$ (Disput Compression API 617/API 618/ISO 10439) Compression API 617/API 618/ISO 10439	e g W-P+T • Start Type Node (1) Suction, Top Node (3) Discharge, Te aut1 est2 Summary Los	DN, mm 200 \$ 200 ds 250.91	Frad, N F -15918 1440505 1424587	eed the Show Er cir, N Florig 2007 -2320 -173 0 2734 -2825	N         FR, N           0         30062           1440505           14424833	7 Miad, N-m -2577.83 0 -2577.83	Mcir, N-m 11010.28 22853.07	Micing, N-m 8677.81 28.39 15245.88	MR, N-m 14253.98 28.99 27396.16	Sum Note 430 1 14454

Added new object "Turbine NEMA SM23/API 611/API 612", allows to automatically model the steam turbines, consider thermal movements of the nozzles, checks allowable loads using NEMA SM23, API 611, API 612, ISO 10437

TOTAL OF THE OFFICE								
Name								
Material <mark>o</mark> f Compress	ior		20			•		
Manufacturing Tech	nology		Sear	nless		•		
Temperature of Com	pressor	L	100		°C			
Factor for Allowable	Loads		I LINCEPP	1				
Shaft Axis	x	•						
Center of Compresso	r Coordinate t	from Node	_		2			
control of compresse		inom nous			2			
DX 0 mm	DY 500	) n	nm	DZ C		mn		
Remove Restraints f	or Hanger Se	lection						
Don't Remove	•							
Discharge Node 3 • S	et Manual Lo	ads						
Discharge Node 3	et Manual Lo	ads						
Discharge Node 3   Control S Additional Nozzle 1 0  Control S	iet Manual Lo iet Manual Lo	ads						
Discharge Node 3	iet Manual Lo iet Manual Lo	ads						
Discharge Node 3	iet Manual Lo iet Manual Lo iet Manual Lo	ads ads						



Added new object "Other Pump", allows to automatically model the pumps, consider thermal movements of the nozzles, checks allowable loads

	ne						
Materia	l of Pump			20	•		
Manufa	cturing Tech	nology	1	Seamless	-		
Temper	ature of Pum	p	L	100	°C		
Shaft A	xis	x	▼ X: 0	۰	Y: 90 °		
Pump C	enter Coordi	nates from No	ode	3	-		
DX		DY		DZ			
0	mm	500	mm	0	mm		
Remove	e Restraints f	or Hanger Se	election				
Vertica	al	- 🗆 X	🗌 Y 🖂	Z 🗌 RX	RY F	RZ	
Suction	n Node	FR		MR			
3	-	N		Nm			
		1		1			
FX	FY	FZ	МХ	MY	MZ		
N	N	N	Nm	Nm	Nm		
1	1	1	1	1	1		
Discha	rge Node	50		MD			
1	-	N		N <sup>m</sup>			
		1		1			
FX	FY	FZ	MX	MY	MZ		
N	N	N	Nm	N·m	Nm		
1	1	1	1	1	1		
		OK		Cancel	Help		
					K		



Added new object "Fired Heater API 560/ISO 13705", allows to automatically model the fired heaters, consider thermal movements of the nozzles, checks allowable loads using API 560 and ISO 13705



Added new object "Air cooled Heat Exchanger API 661/ISO 13706", allows to automatically model the air cooled heat exchangers, consider thermal movements of the nozzles, checks allowable loads using API 661 and ISO 13706



#### Table 4 — Maximum Allowable Nozzle Loads

Nozzie Size		Momenta Nim (fillal)		Forces N (Br)					
DN (NPS)	M	М,	$M_{c}$	$F_v$	Ε,	$F_t$			
40 (1%)	110 (88)	150 (110)	110 (80)	670 (150)	1020 (230)	670 (150)			
50(2)	150 (110)	240 (180)	150 (110)	1020 (230)	1330 (300)	1020 (230)			
60 (3)	410 (300)	610 (450)	410 (300)	2000 (450)	1690 (360)	2000 (450)			
100 (4)	R10 (600)	1220 (900)	810 (600)	3340 (750)	2670 (600)	3340 (750)			
150 (6)	2140 (1560)	3050 (2250)	1630 (1200)	4000 (900)	5030 (1130)	5030 (1130)			
200 (8)	3050 (2250)	6100 (4500)	2040 (1650)	5890 (1280)	13,340 (3000)	8010 (1900)			
250 (10)	4070 (3000)	6100 (4500)	2550 (1880)	6670 (1500)	13,340 (3000)	10,610 (2250)			
300 (12)	5060 (3750)	6100 (4500)	3050 (2250)	8360 (1880)	13,340 (3000)	13,340 (3000)			
350 (14)	6100 (4500)	7120 (\$250)	3570 (2630)	10,010 (2250)	16,680 (3750)	16,680 (3750)			

#### lues from 7.1.10.2

7.1.10.2 The design of each fixed or floating header, the design of the connections of fixed headers to side frames, and the design of other support members shall ensure that the simultaneous application (sum) of all nozzle loadings on a single header obser not cause any damage. The components of the nozzle loadings on a single header shall not exceed the following values:

- Mg 6100 N m (4500 ft lbf
- M, B130 N m (6000 ft bit
- M2 4070 N m (3000 ft lbf)
- F<sub>1</sub> 10,010 N (2250 8/f)
- Fp 20,020 (4500 RM)
- F: 16,680 (3750 lbf)

am 7.1.10.2 multiplied by 3

7.1.10.3 The total of all nozzle loads on one multi-bundle bay shall not exceed three times that allowed for a single header.



Added new object Untied Expansion Joint and database of Untied Expansion Joints, allows to specify the axial, rotational, shear and torsion flexibility and automatically checks the individual and combined allowable deformations. No need to manually model it using nonstandard expansion joint any more

Ctrl+H



/ 🛄 Input 🐵	🍺 Def exp 🛭 🕻	3											
Operating Mode		Load Case	•			Axis							
1 '操作模式' (0)	•	Operating	W+P+T •			Local axis (De	esign/Allowable) 🔻	?					
Node Number	Туре		Local axis	Axial, (mm)	Allowable, (mm)	Shear, (mm)	Allowable, (mm)	Angular, (°)	Allowable, (°)	Torsion, (°)	Allowable, (°)	Summary	Notes
12	Untied Expansi	on Joint	Pipe 3 - 12	2.41	50	1.22	15	9.59131	10	-2.05119	No	1.09	1
13	Torsion Expansi	ion Joint	Pipe 5 - 13	0	No	0	No	0	No	13.9229	51.5662	0.27	
15	Torsion Expansi	ion Joint	Pipe 7 - 15	0	No	0	No	0	No	10.1299	51.5662	0.20	
21	Torsion Expansi	ion Joint	Pipe 19 - 21	0	No	0	No	0	No	-4.36021	51.5662	0.08	
4													



Added new object Torsion Expansion Joint and database of torsion expansion joints, automatically model torsion friction (friction moment) and checks allowable rotation angle



Added new fitness-for-service abilities: "Plane Flaw" object and "Volumetric Flaw" object, allows to model the plain flaw on the pipe, bend, tee and check the stresses in the flaws





Out	tput <u>W</u> indow <u>H</u> elp		
o <mark>∎ ∳0 os</mark> o	<u>P</u> iping Stress Insulation Stress Seismic Stress (Aboveground) Flaw Stress		
<b>≪</b> €∎	<u>L</u> oad and Displacement in Restraints <u>R</u> estraint Loads Nozzle and Equipment Loads		M
▲ ₩ ₽₽	<u>D</u> isplacements Expansion Joint Def <u>o</u> rmations Iternal <u>F</u> orces & Moments		
?≝ ?≛ ₩	Selected Springs Selected Constant Effort Springs Buckling Check of Pipe <u>W</u> all <u>F</u> lange Leakage Check		*** ***
	Output 3D View Ctrl+H Error & Warning Messages	Plane Flaw Volumetric Flaw	#

🛱 Input 🐵 🍺 Defect Stress 🟮								
eratin 'Main	g Mode mode'(0)	•	Show Equations					
lode	Object	Flaw Type	Defect Stress ,(MPa)		%	Result	Notes	
			calcu- lated	allow- able				
3	Above ground pipe	Plane Flaw	32.55	7.19	452.78	FAILED	1	
4	Above ground pipe	Volumetric Flaw	0.030	0.44	6.85	PASS		
5	Forged Elbow	Volumetric Flaw	0.046	0.44	10.40	PASS		
6	Welding Tee	Volumetric Flaw	0.051	0.44	11.51	PASS		
8	Above ground pipe	Volumetric Flaw	pr=2 MPa					
			Ri=103.5 n	nm				
			Ro=109.5	mm				
			T=6 mm					
			F=6730.70	29 kgf				
			M=387.37	067 kgf·cm				
			a=1.17041	1				
		c=0.333333						
			b=0.061509					
			σs=221 M	Pa				
			σb=200 M	Pa				
			σ'=(σs+σł	o)/2=210.5 I	MPa			
			PI0=2/(3)/	`0.5*σ'*log	(Ro/Ri)=	1369.7396	28	
			MI0=4σ'*(	Ro^3-Ri^3	)/3=5.73	1620		
			Ac=c*(mii	n(3,a)*b*c)/	1/3=0.0	96146		
			pls=0.95-(	0.85+0.013	a/b)*Ac=	0.844493		
			Pls=pls*Pl	0=1156.734	923			
			mls=cos(1	τ/2 *cb)-c//	2*sin(πb)	=0.96747	5	
			Mls=mls*l	MI0=5.5452	.01			
			pr^2/pls^	2+M^2/MI	s^2=0.02	29895		

ISS PIPING AND ANALYSIS &

Added ability to specify insulation, cladding, and liner layers density and thickness in pipe properties. The ability to choose an insulation weight from the database still exist

Test Pressure	0	MPa					-
Uniform Weight							
Calculate Pipe Weigh	nt Automatically						
Pipe	179.98	kgf/m					
Insulation	L 49.31	kgf/m >					ti, ltc
Fluid	L 14.74	kgf/m		Insulation Thickness	50	mm	1 2
Fluid Density	L 1000	kg/m3		Inculation Donoity	800	ka/m2	
				Insulation Density	000	kg/m5	//4
			L	Cladding Thickness	10	mm	
OK	Cancel	Help		Cladding Density	1500	kg/m3	
			L	Lining Thickness	0	mm	
				Lining Density	o	kg/m3	
						_	D .



Added thumbnails for windows explorer. Now you can preview all piping models right in the explorer before opening the file





Added function "Copy Whole Model". Allows to copy whole piping model as an object into clipboard. After that you can insert this interactive model into any other software like MS WORD, EXCEL etc. You can rotate, pan zoom the model right inside MS WORD

You can add interactive into report in MS Word and send for your customer for review





- Added export to EHS pipe support sizing software. START-PROF can create cad8.dat file
- Added function that save backup copy of the file before each run with date and time stamp. It is available in general settings
- Added automatic compression of START-PROF piping model files (.ctp). Now files become 10 times smaller
- Significantly increased the speed of opening and saving big piping models into file
- Added new topics into Application Guide



### PASS/Start-Prof | Reliability

Full Verification and Validation manual. Added a lot of verification examples, compared to manual calculations and other software.



Each new PASS/START-PROF release is:

- Automatically verified on more than 300 examples with previous versions (quality assurance system)
- Checked manually with group of pipe stress experts (testers)
- Each release pass through 1-3 pipe stress trainings with 10-20 students before official release
- After release, all bugs reported by +2 000 companies of our active users are quickly fixed and new release is provided
- Software is trusted and reliable!



Subscribe our YouTube channel, you will find a lot of PASS/START-PROF training videos

www.youtube.com/passuite





## PASS/Start-Prof | Licensing

#### Configurations/Pricing Options

#### PASS/Start-Prof Complete Advanced

Simulation and sizing for any piping network considering all applicable national codes.

#### PASS/Start-Prof Complete Standard

configuration includes only worldwide popular standarts.

#### PASS/Start-Prof Process Advanced

Simulation and sizing for piping networks based on applicable national codes for process plants as well as for gas and oil transportation systems.

#### PASS/Start-Prof Process Standard

configuration includes only worldwide popular standarts.

PASS/START-PROF PASS/Start-Prof Power Advanced HDPE+FRP Simulation and sizing for any Piping stress analysis of high density polyethylene and/or piping networks based on applicable national codes for fiberglass reinforced plastic power generation piping as piping systems. well as for central heating networks.

#### PASS/Start-Prof Power Standard

configuration includes only worldwide popular standarts.

#### Permanent License at Affordable Price (+1 year maintenance for free!)

- Maintenance Renew 1 Year: 25%
- Annual License: 40%
- Semi-Annual License: 25%



#### **Complete Standard Process Standard** HOPE+FRP **Power Standard** Code Complete Process Power Advanced (40% discount) (40% discount) (40% discount) (40% discount) Advanced Advanced ISO 14692 4 0 **HDPE Piping** 4 4 4 ASME B31.1 ~ 4 V 4 ASME B31.3 ¥. V v 4 v ~ ASME B31.4 V 4 ASME B31.5 ¥. V. ~ 4 $\mathbf{v}$ ¥. ASME B31.8 4 V v 4 V. 4 ¥. 4 ¥. ASME B31.9 ~ EN 13480 v V V 4 4 4 4 4 GB 50316 4 1 2 4 V 4 4 G8/T 20801 4 4 GB 50251 4 4 4 4 4 4 GB 50253 4 4 4 1 DL/T 5366 ~ 4 CUJ/T 81 4 4 RD 10-249-98 ~ 4 **GOST R 55596** GOST 32388 4 4 4 SNIP 2.05.06-1 85 SP 36.13330 ~ 4

#### **Configurations Comparison**

### PASS/Start-Prof | Resources

Subscribe to our Social Media to Learn More!

- Web site: <u>www.passuite.com</u>
- YouTube Channel: <u>www.youtube.com/passuite</u>
- LinkedIn: <a href="http://www.linkedin.com/company/passuite/">www.linkedin.com/company/passuite/</a>
- Facebook: <u>www.facebook.com/PASSuite</u>
- Twitter: <u>twitter.com/passuitecom</u>
- More than 50 articles about pipe stress analysis and PASS/START-PROF features <u>https://whatispiping.com/category/start-prof</u>



### PASS/Start-Prof | Resources

Subscribe our YouTube channel, you will find a lot of PASS/START-PROF training videos

www.youtube.com/passuite





P: +7 495 225 94 32 F: +7 495 368 50 65 E: sales@passuite.com W: www.passuite.com



# Thank YOU!